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# Isogeometric Analysis of Structures Local Treatment of Constraints

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- This lecture presents:
  - Summary of current IGA capabilities using FEAP.
  - Some methods to treat constraints in IGA.
  - Application to thin structures.
    - \* Straight and curved beam
    - \* Comparison with rotation free thin shell (Kiendl, *et al.*, CMAME, vol 198, pp 3902ff, 2009)
  - Graphics for NURBS.

## **Brief Overview of FEAP**

- FEAP Finite Element Analysis Program.
- <u>Research and educational</u> software package developed at University of California, Berkeley.
- Includes <u>element library</u>: Solids, Thermal, Frames, Plates, Membranes & Shells.
- Elements for both small and large deformation analysis.
- <u>Material library</u> for: Elastic, visco-elastic, elasto-plastic, ...
- Solution algorithms by command language statements.
- Screen and hard copy plotting options.
- User module interfaces for <u>elements</u>, <u>meshing</u>, <u>solution</u>, <u>plots</u>.

Used for NURBS and T-spline isogeometric solutions.

- Isogeometric models described by:
  - Knot vectors (open)
  - Control points and weights
  - Tensor product NURBS (Non-Uniform Rational B-Splines) or T-splines
- IGA elements in *FEAP* 
  - Displacement formulations (all FEAP solid elements work)
  - Mixed  $\mathbf{u} p \overline{J}$  formulation (2-d finite deformation only)
  - Thin rotation-free thin shell (Kiendl et al.)
  - Some 'user module' elements.
- Boundary conditions (only restricted types, no contact)
- Most solution options work (transient, eigenpairs, etc.)



$$B_{i,p}(\xi) = \frac{\xi - \xi_i}{\xi_{i+p} - \xi_i} B_{i,p-1}(\xi) + \frac{\xi_{i+p+1} - \xi}{\xi_{i+p+1} - \xi_{i+1}} B_{i+1,p-1}(\xi)$$

where n = m - p - 1 for open knots. e.g.,  $\Xi = \begin{bmatrix} 0, 0, 0, 0, \frac{1}{4}, \frac{1}{2}, \frac{3}{4}, 1, 1, 1, 1, 1 \end{bmatrix}$ , p = 3 for figure.

• Non-Uniform Rational B-spline (NURBS) defined by

$$N_{i,p}(\xi) = \frac{B_{i,p}(\xi) w_i}{\sum_{j=1}^{n} B_{j,p}(\xi) w_j}$$

where  $w_i$  are set of n weights defining shape of NURBS.

- Appropriate values of  $w_i$  permit description different types of curves; e.g., conic surfaces in addition to polynomials
  - Polynomials in  $\xi$ : Weights  $w_i = 1$  yields:  $N_{i,p}(\xi) = B_{i,p}(\xi)$
  - Circular arc :  $\mathbf{x} = N_{1,2}(\xi) \, \tilde{\mathbf{x}}_1 + N_{2,2}(\xi) \, \tilde{\mathbf{x}}_2 + N_{3,2}(\xi) \, \tilde{\mathbf{x}}_3$



#### Analysis procedure in FEAP

- Define <u>coarse</u> set of control points, knots, 1-d knot-point list, side-patch description:
- Example: Curved beam input <u>NURBS</u>, <u>knot</u>, <u>side</u> & <u>block</u>



- Need to add material properties, loading and boundary conditions.
   Use standard FEAP commands for most.
- Analysis requires degree elevation and knot insertion.
- Example: Elevate order to cubic intepolation

```
BATCh
ELEVate INITialize
ELEVate KNOT 1 1
ELEVate END
END
```

• Knot vector now:  $\Xi = (0, 0, 0, 0, 1, 1, 1, 1)$ 



Add knots: *k*-refinement in circumferential direction



Add knots: *k*-refinement in FEAP

• Knot insertion performed as

BATCh INSErt INITialize INSErt KNOT knum uu rr INSErt END END

where uu knot value & rr number times to repeat.

- Each knot insertion lowers continuity by one order.
- Degree elevation and knot insertion create mesh for analysis.
- Elements defined on knot intervals.

• Typical displacement formulation element module



• IGA modifies **quadrature** and **interpolation** modules.

- Displacement methods for IGA standard.
- Solution of structures often involves constraints (shear-bending, membrane-bending, etc.)
- Known methods to treat constraints
  - Reduced integration
  - Mixed variational methods
    - \* Hellinger-Reissner:  $\mathbf{u} \boldsymbol{\sigma}$
    - \* Veubeke-Hu-Washizu:  $\mathbf{u} \boldsymbol{\sigma} \boldsymbol{\epsilon}$
  - Stabilized methods: GLS, etc.
  - Discrete strain gap: (DSG)
- Bézier extraction provides option for analysis.

#### **Bézier Extraction Form for Elements**

- Extraction converts B-splines & NURBS to Bézier form.
  - Example: Cubic B-Spline (elements between dotted lines)



• Shape functions: extraction operator times Bernstein polynomials.

#### **Bézier Extraction Form for Elements**

• For curves: Interpolations in rational form

$$R_a = \frac{w_a B_a(\boldsymbol{\xi})}{W(\boldsymbol{\xi})}$$
 where  $W(\boldsymbol{\xi}) = \sum_b w_b B_b(\boldsymbol{\xi})$ 

where  $w_a$  is a weight for the *a* basis function.

- Permits representation of conics and other curves:  $w_a = 1$  gives polynomial.
- Surfaces and solids use tensor products of  $R_a$  functions.
- NURBS extraction operator form becomes:

$$\mathbf{N}^{e}(\boldsymbol{\xi}) = \mathbf{C}^{e} \mathbf{R}^{e}(\boldsymbol{\xi})$$

- Shape function routine given as option to standard FE form.
- Ref: M.J. Borden et al.: IJNME, vol. 87, pp 15-47, 2011.

#### **Bézier Extraction Form for Elements**

• Coordinate interpolation with extraction operator:

$$\mathbf{x}(\boldsymbol{\xi}) = \sum_{\substack{b=1 \\ \text{Bézier}}}^{m} \hat{\mathbf{x}}_{b} R_{b}(\boldsymbol{\xi}) = \sum_{\substack{b=1 \\ a=1}}^{m} \sum_{\substack{a=1 \\ \hat{\mathbf{x}}_{b}}}^{n} \tilde{\mathbf{x}}_{a} \mathbf{C}_{ab} R_{b}(\boldsymbol{\xi}) = \sum_{\substack{a=1 \\ na=1}}^{n} \tilde{\mathbf{x}}_{a} \sum_{\substack{b=1 \\ Na(\boldsymbol{\xi})}}^{m} \mathbf{C}_{ab} R_{b}(\boldsymbol{\xi})$$

$$= \sum_{\substack{a=1 \\ na=1}}^{n} \tilde{\mathbf{x}}_{a} N_{a}(\boldsymbol{\xi}) \qquad \text{N.B. } n < m$$
NURBS

where m number of Bézier points and n number of control points.

• Isoparametric form for other variables:

$$\mathbf{u}(\boldsymbol{\xi}) = \sum_{b=1}^{m} \hat{\mathbf{u}}_b R_b(\boldsymbol{\xi}) = \sum_{a=1}^{n} \sum_{b=1}^{m} \tilde{\mathbf{u}}_a \mathbf{C}_{ab} R_b(\boldsymbol{\xi}) = \sum_{a=1}^{n} \tilde{\mathbf{u}}_a N_a(\boldsymbol{\xi})$$

• Permits element development with either NURBS or Bézier form.

• Some implemented IGA user elements:

| ELMT | Description                                        |
|------|----------------------------------------------------|
| 1    | NURBS Euler-Bernoulli beam                         |
| 2    | NURBS 1-d rod.                                     |
| 5    | NURBS & T-spline thin $C^1$ plate                  |
| 6    | NURBS & T-spline thin membrane                     |
| 7    | NURBS & T-spline derivative boundary condition.    |
| 8    | Bending patch for Euler-Bernoulli beam ties.       |
| 18   | Global least squares boundary fit for NURBS region |
| 19   | Local least squares boundary fit for NURBS region  |
| 20   | Follower couple to load thin shell element         |
| 24   | Thin isogeometric non-linear shell (Kiendl et al.) |
| 25   | Thin isogeometric linear shell (Kiendl et al.)     |
| 43   | Linear curved beam with shear deformation.         |

FEAP user elements for NURBS/T-spline solutions.

• Some implemented IGA methods:

| UMACR | Description                                     |
|-------|-------------------------------------------------|
| 6     | Extraction operator for 1-d forms.              |
|       | Command: EXTRact BEZIer n_side                  |
| 8     | Elevate directions of NURBS block.              |
|       | Command: ELEVate knot num inc_order             |
| 9     | Insert knot in directions of NURBS block.       |
|       | Command: INSERT knot num u_val times            |
| 2     | ZZ projection of stress for 2-d T-splines       |
|       | Command: NZZP,,n_mat                            |
| 0     | Elements on edges of NURBS blocks or T-splines. |
|       | Command: EIGA,,dir_i x_i                        |

FEAP commands for NURBS/T-spline solutions.

- Treatment of constraints in structural forms
  - Example: Shear-bending and membrane-bending in curved rods
- Consider curved 2D Timoshenko beam:



• Strain-displacement relations

$$\epsilon = \frac{\partial u_s}{\partial s} - \frac{u_r}{R} \quad ; \quad \gamma = \frac{u_s}{R} + \frac{\partial u_r}{\partial s} + \varphi \quad ; \quad \kappa = \frac{\partial \varphi}{\partial s}$$

• Material constitution (linear elastic)

$$N = EA\epsilon$$
 ;  $V = GA_s\gamma$  ;  $M = EJ\kappa$ 

• Weak form





• Unknowns transformed to global coordinates.

• Curved beam problem:



Data: R = 10; h = 0.0001 to 1; EI = 1000; V = 1; q = 0.1.

• Verification by Kiendl et al. shell: Results for 20-cubic elements with end shear.



• Roundoff high for R/h > 1000

• End shear reduced integration results: 4-elements



• End shear reduced integration forces: 4-elements



• Uniform load reduced integration results: 4-elements



• Reduced integration good for 1-d problems. Need more general form for multi-dimensions.

- Discrete Strain Gap:
  - Method for constraints. Lowers order of interpolation.
     (Bletzinger *et al.*:2000; Koschnick *et al.*: 2005)
  - Replaces locking terms by derivative form of interpolations.
  - Example: Shear strain in straight Timoshenko beam.

$$\gamma = \frac{\mathrm{d}w}{\mathrm{d}x} + \varphi = \sum_{b} \left( \frac{\mathrm{d}N^{b}}{\mathrm{d}x} \, \tilde{w}_{b} + N^{b} \, \tilde{\varphi}_{b} \right) \to \left| \bar{\gamma} = \sum_{a} \frac{\mathrm{d}N^{b}}{\mathrm{d}x} \, \tilde{\gamma}_{a} \right|$$

where

$$\tilde{\gamma}_a = \sum_b \int_{-1}^{\xi_a} \left( \frac{\mathrm{d}N^b}{\mathrm{d}x} \, \tilde{w}_b + N^b \, \tilde{\varphi}_b \right) \, j(\xi) \, \mathrm{d}\xi$$

with  $j(\xi) = dx/d\xi$  and  $\xi_a$  nodal coordinates.

• Used in above references for solids, beams, plates & shells.

- Discrete strain gap (DSG) used with NURBS for beams by Echter & Bischoff (CMAME, vol 199, pp 374ff, 2010).
- Using NURBS basis led to tangent with fully occupied shear terms.
- Using Bézier extraction offers possibility of local approximations.
  - Disadvantage: Continuity between elements only  $C^0$ .
  - Disadvantage: Many more nodal points.
  - Advantage: Offers possible scheme for all  $C^0$  problems.

- Interpolate DSG shear on Bézier element.
- Given Bernstein interpolations:  $-1 \le \xi \le 1$

$$N_1^B = \frac{1}{2^n} (1 - \xi)^n$$
$$N_2^B = \frac{n}{2^n} (1 - \xi)^{n-1} (1 + \xi)$$
...

$$N_n^B = \frac{1}{2^n} (1+\xi)^n$$

• Transform from Bernstein to Lagrange interpolation

$$\begin{cases} N_1^B(\xi) \\ N_2^B(\xi) \\ \dots \\ N_n^B(\xi) \end{cases} = \begin{bmatrix} N_1^B(\xi_1) & N_1^B(\xi_2) & \dots & N_1^B(\xi_n) \\ N_2^B(\xi_1) & N_2^B(\xi_2) & \dots & N_2^B(\xi_n) \\ \vdots & \vdots & \ddots & \vdots \\ N_n^B(\xi_1) & N_n^B(\xi_2) & \dots & N_n^B(\xi_n) \end{bmatrix} \begin{cases} N_1^L(\xi) \\ N_2^L(\xi) \\ \dots \\ N_n^L(\xi) \end{cases}$$

where  $\xi_a$  are positions of Lagrange nodes.

• End shear DSG results: 4-elements



• End shear DSG force results: 4-elements, h = 0.01



• DSG expensive compared to reduced integration Provides method that works for multi-dimensions

• Uniform load DSG results: 4-elements



#### **Closure:**

- Described aspects of implementation of IGA into FE software.
- Considered Bézier extraction form for element development.
- Considered issue of locking in structural elements.
- For 1-d Bézier elements reduced integration adequate.
- DSG offers possibility of treatment in multi-dimensions.
- Displacement form adequate for large *p*!